

HabEx Optical Telescope Assembly

H. Philip Stahl

Purpose

- Introduce candidate optical telescope assembly (OTA) architectures
- Illustrate design/analysis process

Agenda

- Definitions, Specification & Assumptions
- 4-meter Monolithic Mirror Concept
- 6.5-meter Segmented Mirror Concept

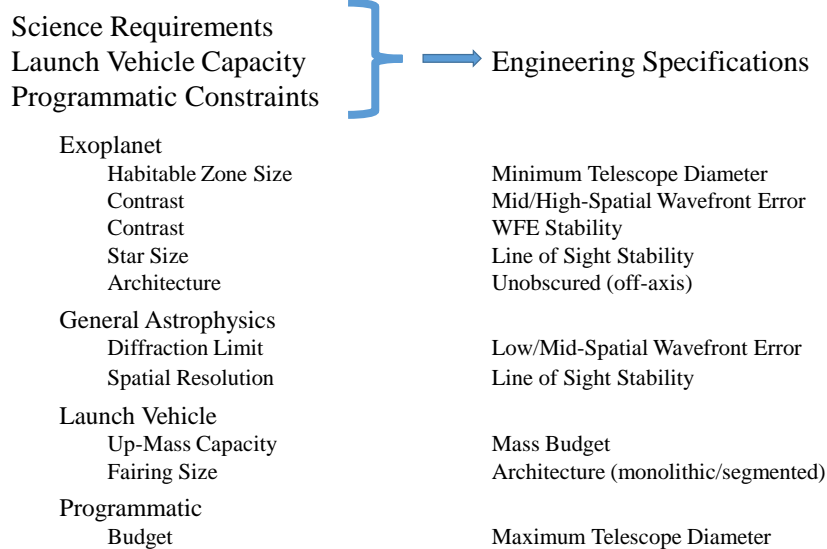
Definitions, Specifications & Assumptions

Optical Telescope Assembly (OTA)

Optical Telescope Assembly (OTA) is defined to consist of:

- Primary Mirror Assembly
 - Primary Mirror Substrate
 - Primary Mirror Struts
 - Primary Mirror Truss Structure
- Secondary Mirror Assembly
 - Secondary Mirror Substrate
 - Secondary Mirror Struts
 - Secondary Mirror Truss Structure
- OTA Structure
 - Connects PMA to SMA and houses Science Instruments (SI).
- OTA Light Tube Baffle

OTA Specification



Design Assumptions

Mission will have an Internal Coronagraph which requires:

- Unobscured Aperture – off-axis
- Stable Wavefront.

General Astrophysics:

- 500 nm diffraction limit requires no development effort

Launch Vehicle

- SLS will exist. Therefore, for ‘baseline’ design mass and volume constraints are secondary to wavefront stability.
- ‘Backup’ designs will be considered for EELV.

The Most important Design Constraints are:

- Line of Sight Stability
- Wavefront Stability

DRAFT OTA Specifications

Architecture	Unobscured Off-Axis F/2.5 TMA	
Aperture Diameter	4-meters Monolithic (Minimum) 6.5-meters Segmented (Maximum)	
Mass Budget	< 10,000 kg (nominal – assumed met)	
Line of Sight Stability	< ~5 milli-arc-second	
Diffraction Limit	500 nm (assumed to be achievable)	
Wavefront Error	36 nm rms Total (assumed achievable)	
Primary Mirror (cpd = cycles/diameter)	Total SFE	< 8.0 nm rms
	Low-Order (< 3 cpd)	< 5.6 nm rms
	Mid-Spatial (3 to 60 cpd)	< 5.6 nm rms
	High-Spatial (>60 cpd)	< 0.6 nm rms
	Roughness	< 0.2 nm rms
Wavefront Stability	Architecture and Coronagraph Specific	

DRAFT OTA Line of Sight Stability

PSF ($2.44\lambda/D$ full-angle) at 500 nm

4-m	~300 nano-radian ~ 60 mas
6.5-m	~200 nano-radian ~ 40 mas

LOS Jitter < 5 mas (1/8th of 40 mas)



DRAFT OTA Wavefront Stability

From Garreth Ruane and Dimitri Mawet, RMS wavefront tolerances for vector vortex coronagraphs over $2.5\text{-}3.5 \lambda/D$.



Aberration	Indices		Allowable RMS error (nm) per mode		
	n	m	charge 6	charge 8	charge 10
Tip-tilt	1	± 1	5.5	18	31
Defocus	2	0	4.6	15	36
Astigmatism	2	± 2	0.36	1.0	4.6
Coma	3	± 1	0.44	0.95	5.5
Spherical	4	0	0.32	0.81	6.7
Trefoil	3	± 3	0.0065	0.35	0.71

From Stahl, Stahl and Shaklan, PV wavefront tolerances for a 4th order radial coronagraph over $1.5\text{-}2.5 \lambda/D$.

Aberration	Tolerance	units
Tip/Tilt	9.6	nm
Power	1.1	nm
Astigmatism	6.8	nm
Seidel Coma	0.84	nm
Spherical	0.3	nm
Trefoil	6.0	nm
Hexafoil	9.6	nm

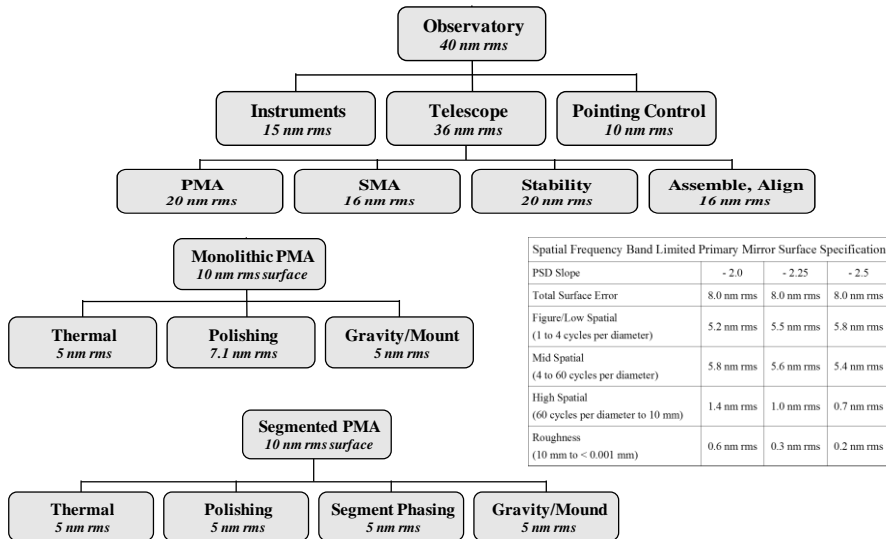
Design for Stability

Wavefront and Line of Sight Stability has design consequences.

- Mechanical
 - Secondary Mirror Support Structure Dynamic Response
 - Primary Mirror Dynamic Response
 - Passive/Active Vibration Isolation
 - Passive/Active Dampening/Control
- Thermal
 - PM & SM Mirror CTE
 - Structure CTE
 - Passive Thermal Isolation
 - Active Thermal Control

BACKUP: Diffraction Limit WFE

Diffraction Limit of 500 requires total system WFE ~ 38 nm rms

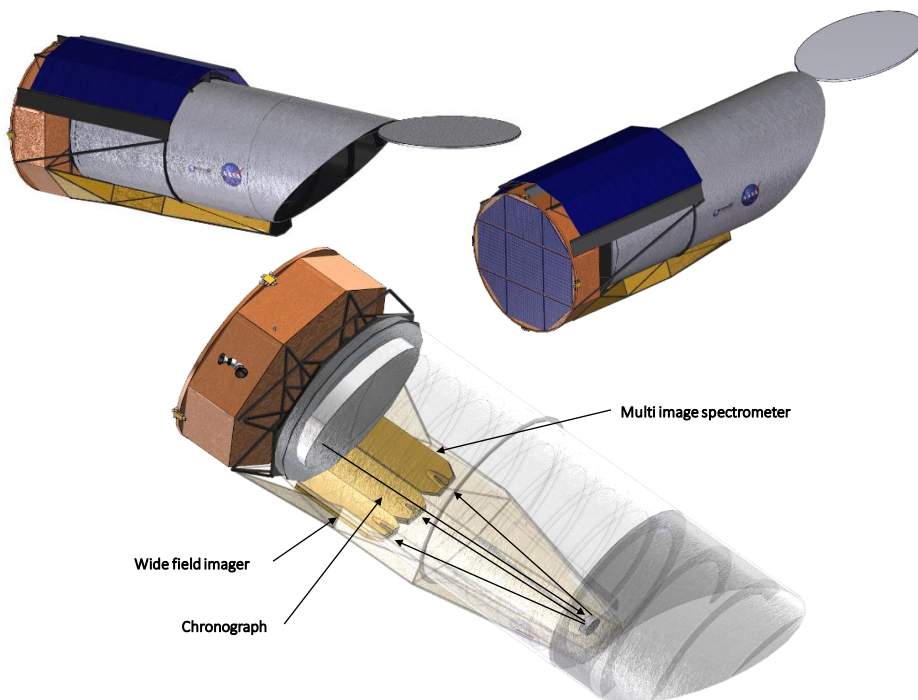
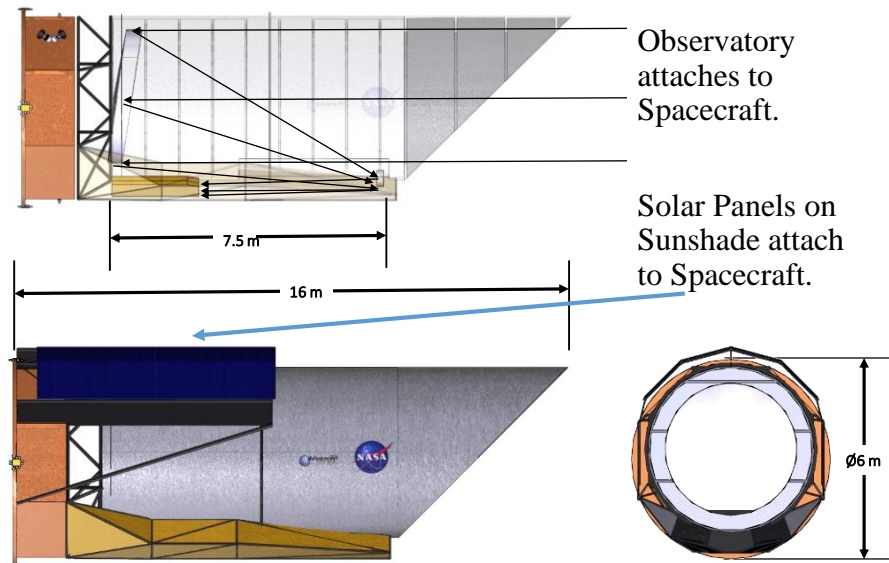


4-meter Monolithic Concept

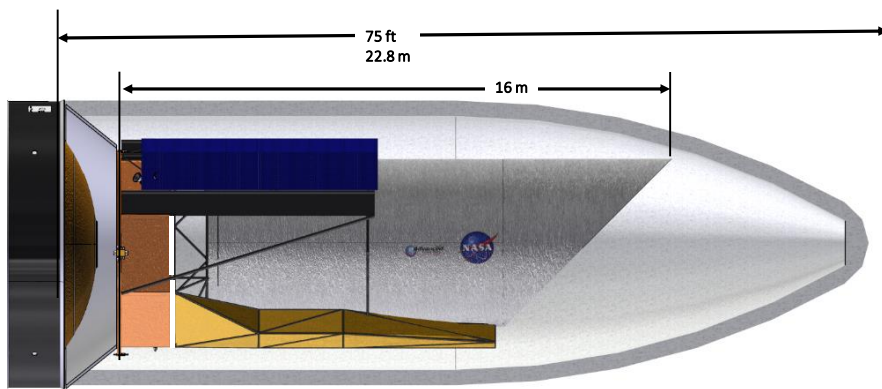
Concept
Tolerances
Structural Design
Primary Mirror Design

HabEx 4-m Off-Axis Concept

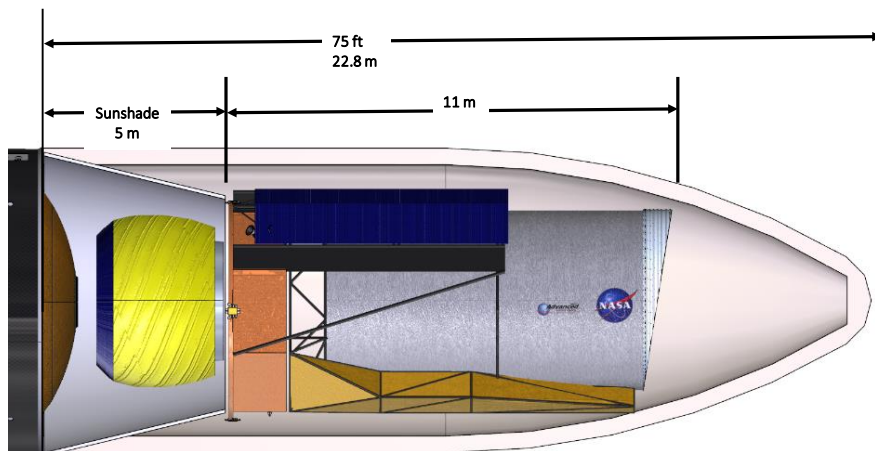
Observatory = OTA (PM/SM/Tube) & Science Instruments.



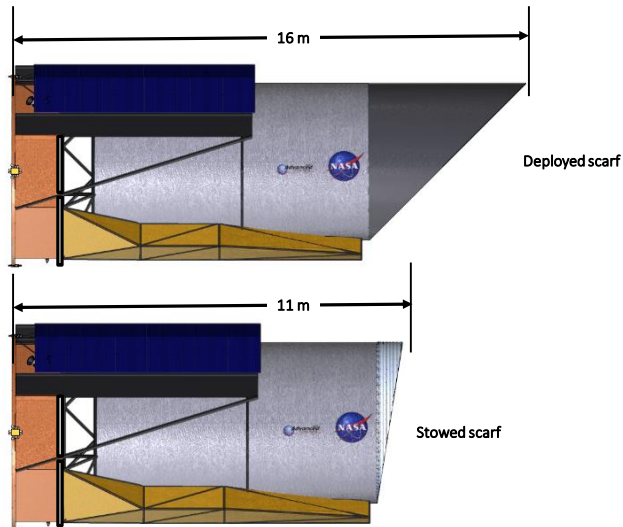
Launch Configuration – no deployments



Launch Configuration – Star-Shade below

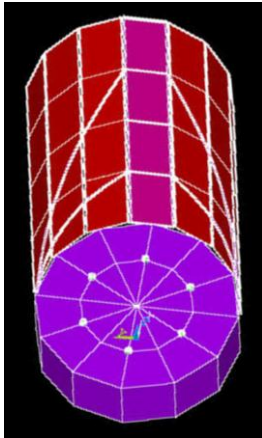


Deployed Forward Scarf

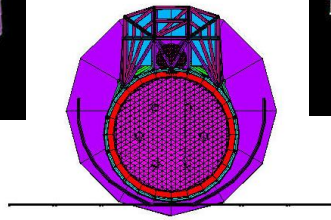
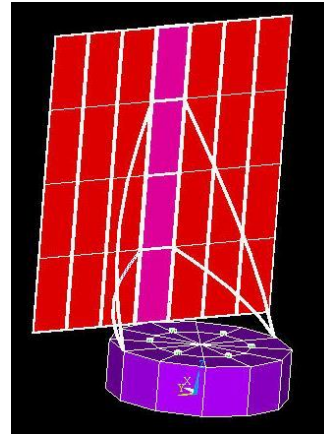


Spacecraft & Sun Shade / Solar Panels

Before Deployment



After Deployment



Tolerances

LOS & WFE Stability drive Mechanical Tolerances

Wavefront Stability

From Garreth Ruane and Dimitri Mawet, RMS wavefront tolerances for vector vortex coronagraphs over $2.5\text{-}3.5 \lambda/D$.

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Astigmatism	6.8	nm
Seidel Coma	0.84	nm
Spherical	0.3	nm
Trefoil	6.0	nm
Hexafoil	9.6	nm

BACKUP: Wavefront Stability Tolerance

Wavefront Stability is driven by Coronagraph Contrast Leakage.

Wavefront Errors (WFE) are caused by:

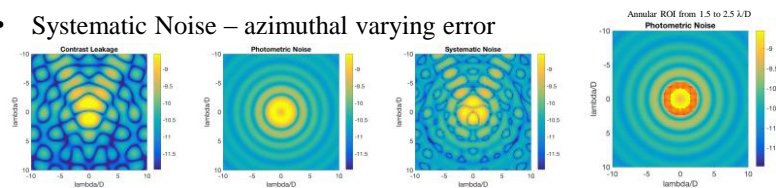
- OTA response to mechanical disturbances
- OTA response to thermal perturbation.

Following the definitions and methodology published by:

Stuart B. Shaklan, Luis Marchen, John Krist and Mayer Rud, “Stability error budget for an aggressive coronagraph on a 3.8m telescope”, SPIE Proceedings 8151, 2011.

Contrast leakage decomposed into radial & azimuthal components

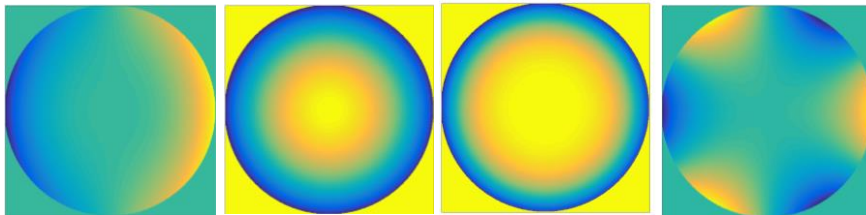
- Photometric Noise – time and spatial averaged radial
- Systematic Noise – azimuthal varying error



BACKUP: Wavefront Stability Tolerance

For Monolithic Aperture, the primary WFEs in response to mechanical stimuli are:

- Alignment Error from motion of PM relative to SM
 - Lateral Displacement produces Seidel Coma (Zernike Coma & Tilt)
 - Longitudinal Displacement produce Power and Spherical
- Bending of PM reacting against its mount - Trefoil



PM to SM Decenter:
Coma & Tilt

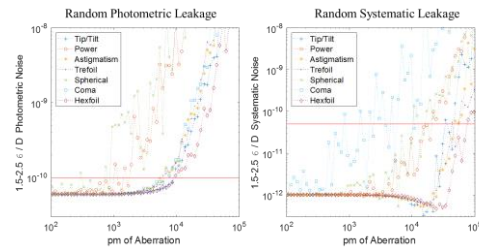
PM to SM Despace: Power and Spherical

PM Mount: Trefoil

BACKUP: Coronagraph Contrast Leakage at 1.5-2.5 λ/D

Tolerance for Monolithic with 4 th Order Radial Coronagraph				
Aberration	Random	Static	Sinusoidal	units
Zernike Tip/Tilt	9,600	9,600		pm
Seidel Power	1,100	190		pm
Zernike Astigmatism	6,800	6,800		pm
Seidel Coma	840	260		pm
Seidel Spherical	300	73		pm
Zernike Trefoil	6,000	6,800		pm
Zernike Hexafoil	9,600	11,000		pm

1.5-2.5 λ/D	WFE (pm) for 1×10^{-10} Photometric Noise	WFE (pm) for 5×10^{-11} Systematic Noise
Tip / Tilt	9,600	35,000
Power	1,100	22,000
Astigmatism	6,800	49,000
Trefoil	6,000	44,000
Hexafoil	9,600	78,000
Spherical	300	11,000
Seidel Coma	6,800	840



Wavefront Error (WFE) Stability Specification

Optical Design WFE Alignment Sensitivity

F/2.5 Design											
Alignment	Tolerance	Units	Z1	Z2	Z3	Z4	Z5	Z6	Z7	Z8	Units
PM X-Tilt	1	micro-degree	431	15	15	18	-648	214	0	0	pm PV
PM Y-Tilt	1	micro-degree	0	444	7	668	0	0	221	2	pm PV
SM X-Tilt	1	micro-degree	35	0	0	0	53	18	0	0	pm PV
SM Y-Tilt	1	micro-degree	0	84	585	91	0	0	42	0	pm PV
SM X-Decenter	10	nanometer	25	0	0	37	0	0	12	0	pm PV
SM Y-Decenter	10	nanometer	0	22	37	0	36	11	0	0	pm PV
SM Z-Despace	10	nanometers	0	12	146	0	9	6	0	3	pm PV

Alignment specification to achieve WFE Stability

Rigid Body Tolerance for F/2.5 Optical Design					
Alignment	from 1.5 to 2.5 λ/D	from 2.5 to 3.5 λ/D			Units
	4 th Order Radial	VVC6	VVC8	VVC10	
PM X-Tilt	10	10	25	180	nano-radians
PM Y-Tilt	10	10	25	180	nano-radians
SM X-Tilt	100	60	250	500	nano-radians
SM Y-Tilt	100	60	250	500	nano-radians
SM X-Decenter	100	100	250	1000	nanometers
SM Y-Decenter	100	100	250	1000	nanometers
SM Z-Despace	25	500	1000	1000	nanometers

- 4th order Radial tolerance driven by coma and defocus sensitivity.
- VVC tolerances driven by astigmatism sensitivity

Line of Sight (LOS) Specification

Optical Design LOS Alignment Sensitivity

Alignment	Tolerance	Units	F/2.5	F/2	F/1.5	Units
PM X-Tilt	1	micro-degree	35.2	35.4	35.6	nano-radian
PM Y-Tilt	1	micro-degree	34.6	34.5	34.2	nano-radian
SM X-Tilt	1	micro-degree	3.93	4.48	5.54	nano-radian
SM Y-Tilt	1	micro-degree	2.87	2.85	2.79	nano-radian
SM X-Decenter	10	nanometer	0.91	1.11	1.45	nano-radian
SM Y-Decenter	10	nanometer	0.89	1.11	1.45	nano-radian

Alignment specification to achieve LOS < 5 mas (24 nrad)

Multiple potential distribution of tolerances

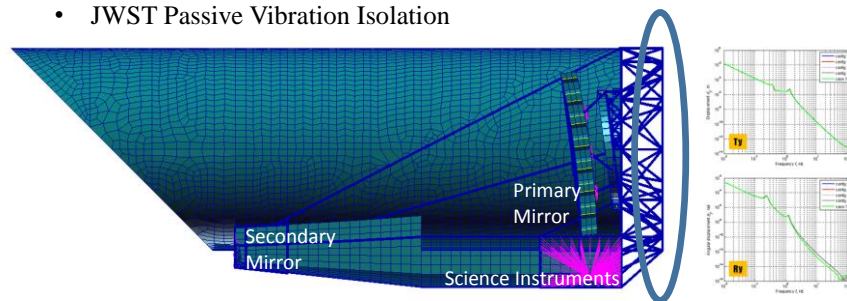
Distribute LOS Tolerance between all Alignment Errors					
Line of Sight Stability	2	4	5	6	mas
PM X-Tilt	2.5	6	8	10	nano-radians
PM Y-Tilt	2.5	6	8	10	nano-radians
SM X-Tilt	7	20	16	15	nano-radians
SM Y-Tilt	7	20	16	15	nano-radians
SM X-Decenter	50	50	50	50	nanometers
SM Y-Decenter	50	50	50	50	nanometers

5 mas LOS Stability is more difficult than WFE Stability

Structure Dynamic Analysis

To determine PM/SM Rigid Body motions, apply at the OTA/Spacecraft interface:

- JWST Reaction Wheel Vibration Specification
- JWST Passive Vibration Isolation



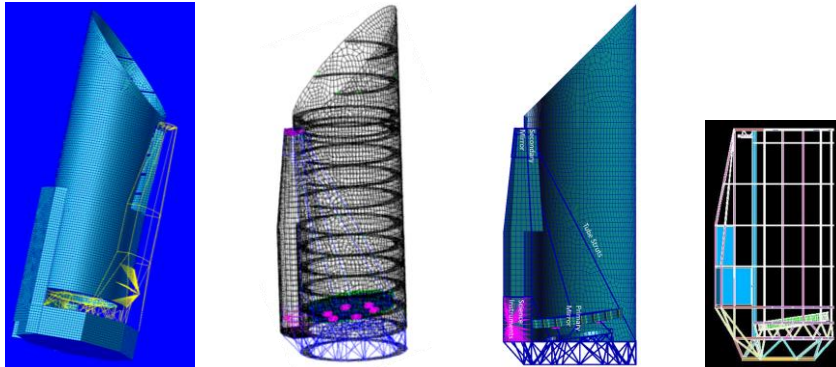
Key Unknowns that impact analysis

- 1) Dimensions of Interface between OTA and Spacecraft
- 2) Amplitude and Location of Science Instrument Mass

Secondary Mirror Support Structure Stability

Studied Four Design Concepts

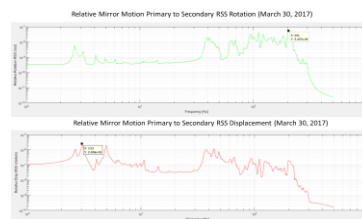
- Free-Standing Tower (~ 10 Hz)
- Tower Attached to Baffle Tube (~20 Hz)
- Tower Attached to Baffle Tube with Struts (~30 Hz)
- HST style Truss Structure



30 Hz Secondary Mirror Structure Dynamic Response

Apply JWST Reaction Wheel Assembly Disturbance Specification at 4 locations in the Spacecraft and calculate rigid body motions of the Primary and Secondary Mirrors with 1% dampening

Rigid Body Alignment Motion			
DOF	RWA Disturbance Only	RWA Disturbance & Isolation	Units
PM/SM Y-Rotation	54		n-rad
PM/SM X-Rotation	44		n-rad
PM/SM X-Despace	910		nm
PM/SM Y-Despace	2490		nm
PM/SM Z-Despace	1000		nm

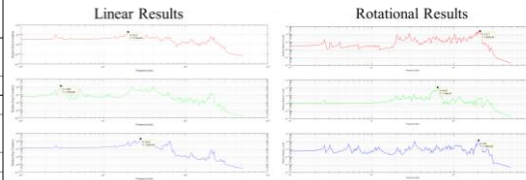


Passive Isolation Required to meet SM Rigid Body Motion Specification						
Alignment	LOS	WFE				Units
		4 th Radial	VVC6	VVC8	VVC10	
PM/SM Y-Rotation	6X	0.4X	0.7X	0.2X	0.1X	nano-radians
PM/SM X-Rotation	5X	0.5X	1X	0.2X	0.1X	nano-radians
PM/SM X-Despace	12X	10X	10X	4X	1X	nanometers
PM/SM Y-Despace	35X	25X	25X	10X	2.5X	nanometers
PM/SM Z-Despace	NA	40X	2X	1X	1X	nanometers

BACKUP: Tower Attached to Baffle Tube (~20 Hz)

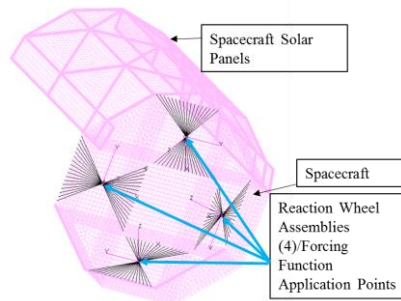
RWA Vibration only, no Passive Isolation, 1% dampening.

Relative Mirror Motion	Max	Description
X (meters)	3.78E-06	De-Center
Y (meters)	1.37E-06	Vector Components
Z (meters)	1.92E-06	De-Space
RSS (meters)	3.78E-06	Net De-Center
RX (Radians)	2.52E-08	LOS Delta
RY (Radians)	1.15E-07	Vector Components
RZ (Radians)	1.50E-08	N/A
RSS (Radians)	1.15E-07	Net LOS Delta



Passive Isolation Required to meet SM Rigid Body Motion Specification						
Alignment	LOS	WFE				Units
		4 th Radial	VVC6	VVC8	VVC10	
SM X-Tilt	12X	0.3X	0.4X	0.1X	0.05X	nano-radians
SM Y-Tilt	3X	1.2X	2X	0.5X	0.2X	nano-radians
SM X-Decenter	50X	40X	40X	15X	4X	nanometers
SM Y-Decenter	20X	15X	15X	6X	1.5X	nanometers
SM Z-Despace	NA	80X	4X	2X	2X	nanometers

BACKUP: Reaction Wheel Assemblies



Reaction Wheel Assemblies (RWA) in common pyramid arrangement providing three axis control with redundancy.

RWA disturbance forces and moments applied locally at grids.

RWA radial force and radial moment disturbances are swept through the 360 degree wheel rotation in order to calculate maximum relative displacement between primary and secondary mirror due to a each wheel.

Enveloped disturbances from each RWA are linearly combined to produce the overall maximum relative displacement between the primary and secondary mirror during three axis control.

BACKUP: Reaction Wheel Assembly Distrubance

3.3.1.6 Wheel Unbalance

After exposure to the environments defined in section 3.2.5 of this specification, the unbalance magnitude of the RWA rotating components shall not exceed the following values:

- Static Unbalance: Less than 1.0 (g-cm) over the operating speed range.
- Dynamic Unbalance: Less than 14.0 (g-cm²) over the operating speed range.
- The peak radial forces and moments produced by the RWA at any operating speed (including resonant conditions) shall not exceed the values listed in the table below:

Peak Radial Disturbance Limits			Including Resonant Conditions		
Parameter	Frequency	Max. Limit	Parameter	Frequency	Max. Limit
Force	0-70 Hz	3.5 N	Torque	0-70 Hz	0.5 N-m
(F _x)	70-210 Hz	0.7 N	(M _x)	70-195 Hz	0.3 N-m
	210-270 Hz	10 N		195-225 Hz	1.5 N-m
	270-500 Hz	0.7 N		225-500 Hz	0.3 N-m

3.3.1.7 Axial Induced Vibration

The peak force (amplitude) produced by the RWA in the direction parallel to the its spin axis shall not exceed 0.2 N within the frequency range 2-200 Hz, when measured at constant speeds that are within the operational speed range and that are free of major structural resonances. The peak axial force produced by the RWA at any operating speed (including resonant conditions) shall not exceed the following limit values:

Frequency Range (in Hz):	0-150	150-195	195-225	225-300	300-500
Axial Force (F _z) Limit:	0.7 N	4.5 N	45 N	4.5 N	0.7 N

RWA Spec Vib Level
from Scott Knight, BATC

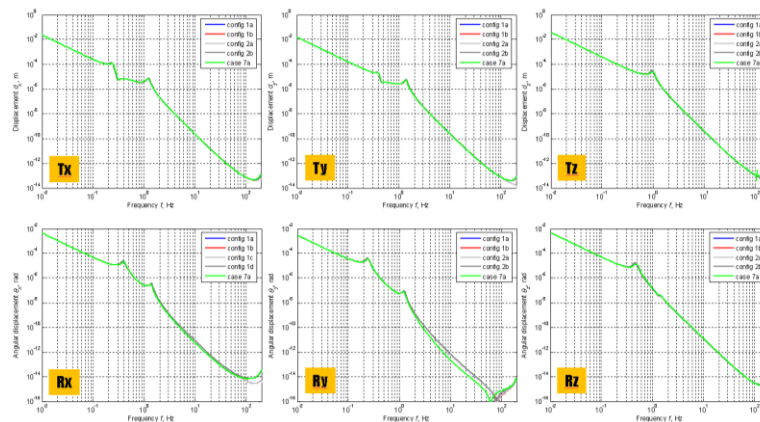
Modal solution is provided by NASTRAN

Boundary conditions to the spacecraft are unconstrained (Free-Free)

Radial force and moment are applied in 10 degree increments around the wheel rotation axis. This results in 144 load cases.

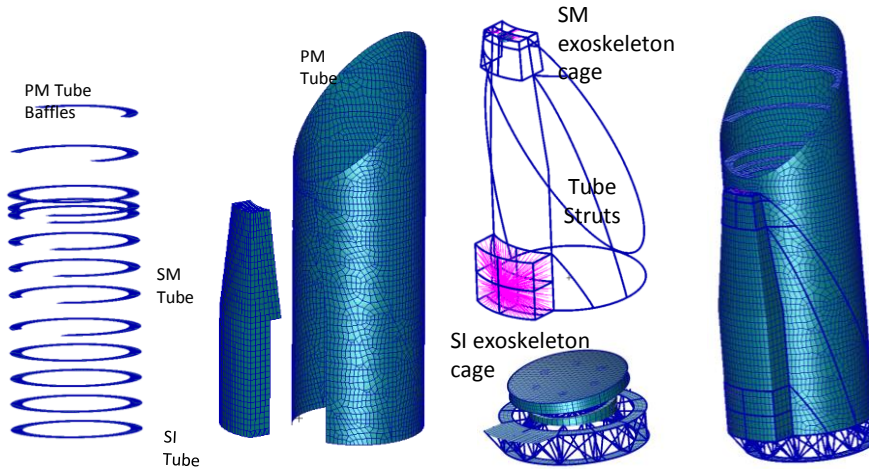
Critical Damping is set at 1%

BACKUP: JWST Passive Vibration Isolation



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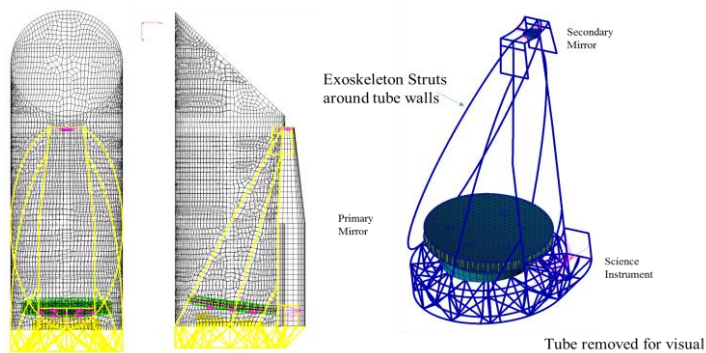
Tower Attached to Baffle Tube with Struts (~30 Hz)



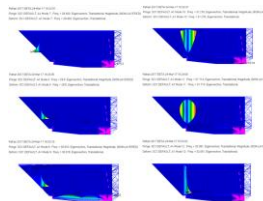
Structural Material assumed to be M55J.

Secondary Support Tower

Exoskeleton provide stiffness without obscuration.



Mode 7 : Freq. = 29.482
 Mode 8 : Freq. = 29.5
 Mode 9 : Freq. = 30.016
 Mode 10 : Freq. = 31.276
 Mode 11 : Freq. = 31.714
 Mode 12 : Freq. = 32.091



Primary Mirror Dynamic Wavefront Error

Dynamic PM WFE arises from two sources:

- Thermal
- Mechanical

Thermal changes produce structural and component motions as a result of material response (bulk CTE and CTE homogeneity)

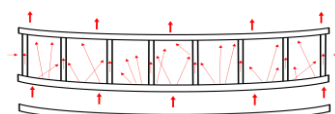
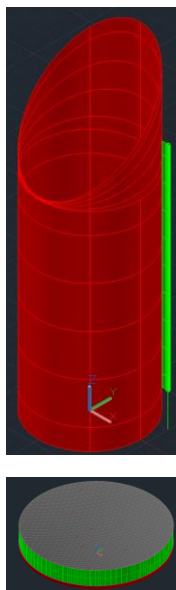
Mechanical Vibrations have a temporal spectrum:

- Specific vibration frequencies induce harmonic modal response.
- All other vibration frequencies cause inertial response.

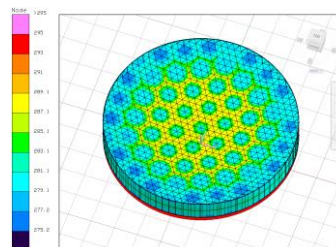
These responses produce structural motions that cause:

- Optical mis-alignment aberrations
- Optical component bending and deformations from mount stress

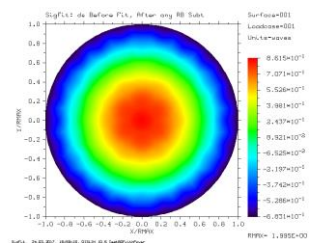
Static Thermal WFE



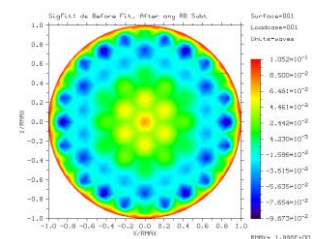
0.5 m thick closed-back ULE mirror
Radial Gradient depends on view factor
and side insulation.



Temperature gradient
Keeping Front Surface > 273K
20C Axial; 10C Radial



SFE from isothermal with defocus
SFE = 977 nm PV; 288 nm RMS

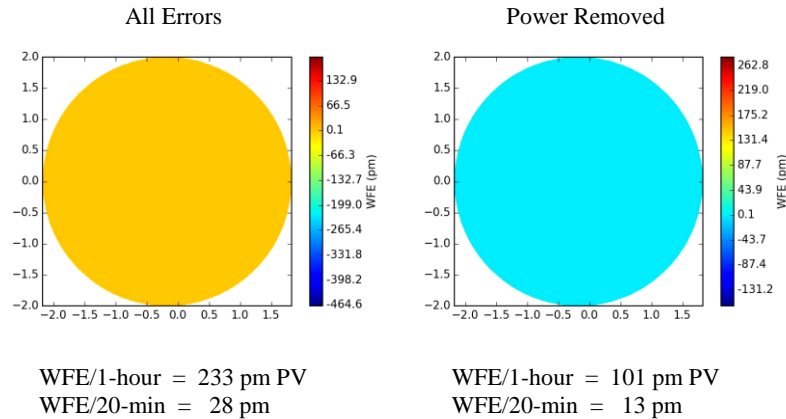


SFE from with defocus removed
SFE = 128 nm PV; 24 nm RMS

Dynamic Thermal WFE Video

Passive Wavefront Error from 1 hour exposure.

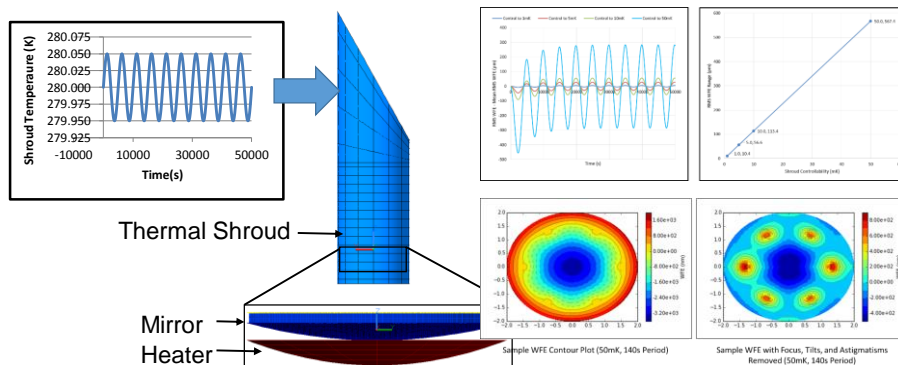
Sun angle changes by 0.0411 degree per hour.



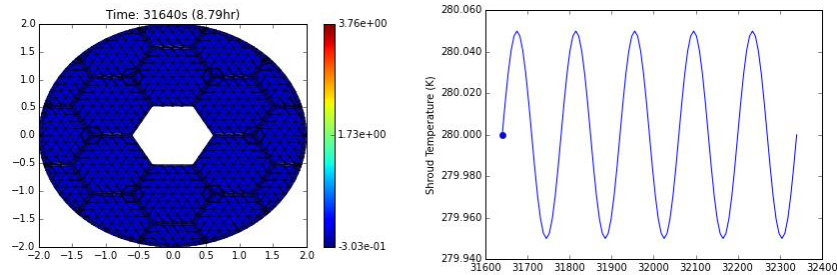
Dynamic Thermal WFE

Primary mirror responds to dynamic external thermal load

Required stability (10 pm per 10 min) can be achieved by controlling the telescope thermal environment.



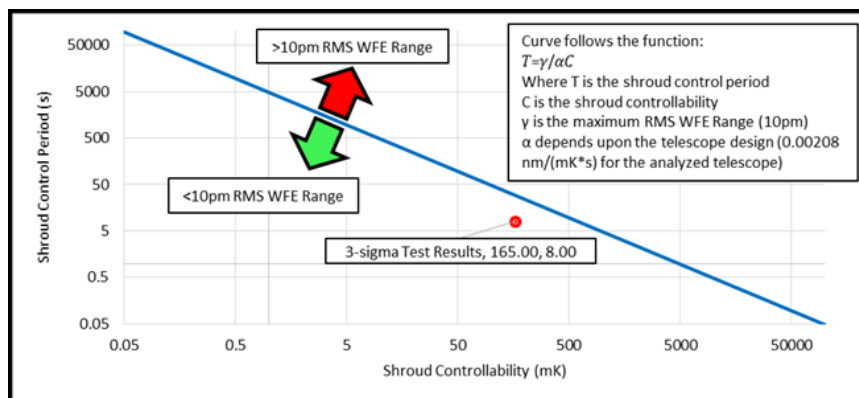
4m Aperture Transient WFE Video



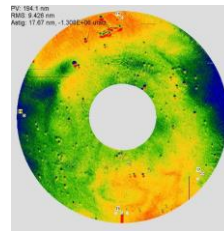
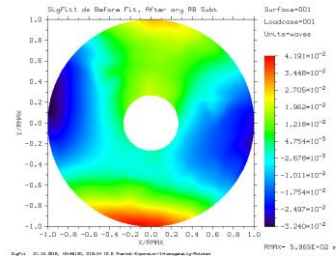
Thermal Stability

The ability to achieve any required wavefront stability depends on:

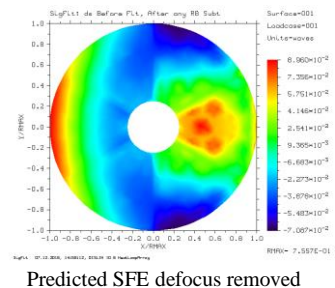
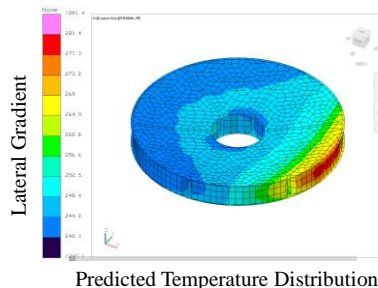
- Mirror Substrate Properties: CTE, Thermal Mass, Conductivity, etc.
- Thermal Environment Controllability
- Control Period.



1.2m Zerodur Predicted vs Measured Thermal WFE



1.5m ULE Predicted Lateral Thermal WFE



Primary Mirror Dynamic Wavefront Error

Dynamic PM WFE arises from two sources:

- Thermal
- Mechanical

Thermal changes produce structural and component motions as a result of material response (bulk CTE and CTE homogeneity)

Mechanical Vibrations have a temporal spectrum:

- Specific vibration frequencies induce harmonic modal response.
- All other vibration frequencies cause inertial response.

These responses produce structural motions that cause:

- Optical mis-alignment aberrations
- Optical component bending and deformations from mount stress

Primary Mirror Dynamic WFE

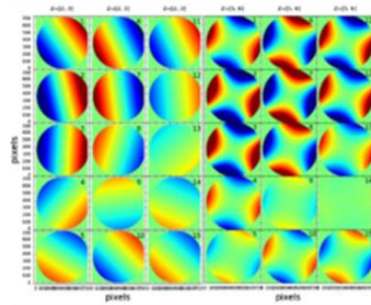
Dynamic WFE depends on the mirror's mounted self-weight deflection (G-sag) and its inertial response function.

G-sag defines the maximum possible WFE for a 1G driving force and a unity response function.

G-sag depends on stiffness of mirror substrate and how it is mounted.

For example, JWST's 220-Hz open-back beryllium primary mirror segments on a 3-point mount have a static horizontal G-sag of approximately 200 nm.

When driven at 87.3 Hz, they have a dynamic Astigmatic WFE of 220 nm per G of driving force.



Saif, et. al., Nanometer level characterization of the James Webb Space Telescope optomechanical systems using high-speed interferometry, Applied Optics, Vol.54, No.13, pg.4295, doi:134285-14, 2015

Primary Mirror Dynamic WFE

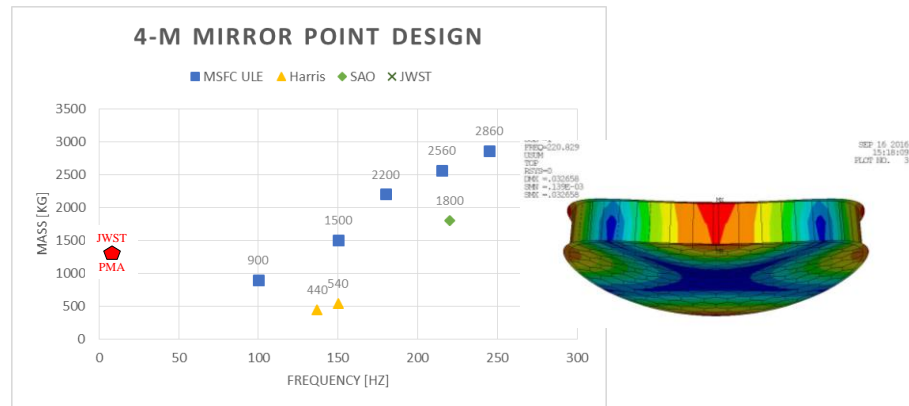
To minimize PM Dynamic WFE:

- Design the PM Substrate to be as stiff as possible
- Consider the Mount stiffness and location.

ULE 4-m Mirror Trade Studies

MSFC explored range of higher mass, more robust designs.

Harris Corporation explored lower limit of mass.



JWST total mass of primary mirror segment assemblies (PMSA) is 700 kg, Total PM Assembly mass is 1250 kg. Individual JWST PM substrates are 220 Hz. Individual PMSA are 40 Hz. Total PMA is 16 Hz.

ULE PM Trade Study: Substrate Stiffness

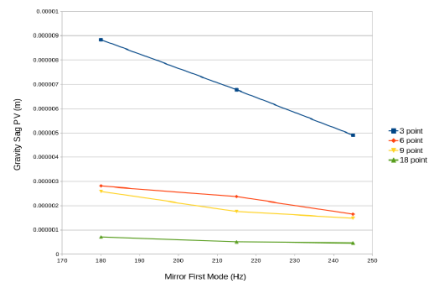
ULE mirrors can be Closed-Back Architectures.

State of Practice Thickness is 35 cm, SOA is 40 cm, 60-cm is developmental.

depth (m)	0.45	0.6	0.75
mass (kg)	1388	1707	1835
cell size (m)	0.167	0.167	0.167
front fs (m)	0.0277	0.0277	0.0277
back fs (m)	0.0231	0.0231	0.0231
1st mode (Hz)	180	215	245

Gravity Sag:

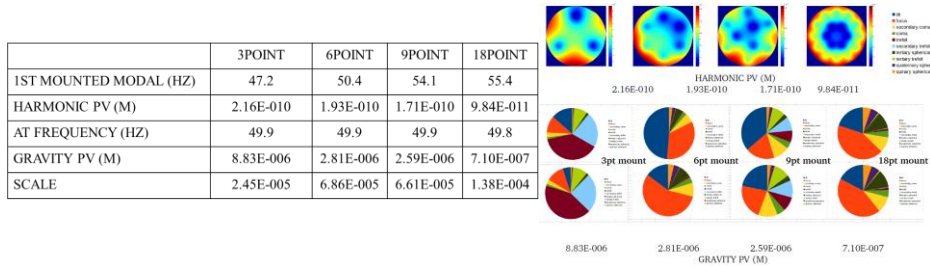
- Stiffer mirror less G-Sag
- More Mount Points less G-Sag



ULE PM Trade Study: Dynamic WFE

Dynamic WFE for 4-m off-axis 180-Hz 1388-kg Mirror as a function of mount support system when excited at 49.9-Hz.

Dynamic WFE amplitude is similar to gravity sag.



Dynamic WFE amplitude goes down with support points.

ZERODUR PM Trade Study: Substrate Stiffness

ZERODUR mirrors are constrained to Open-Back architectures with a maximum thickness of 42 cm.

Trade Studies:

#1: Isogrid (triangular) Pockets

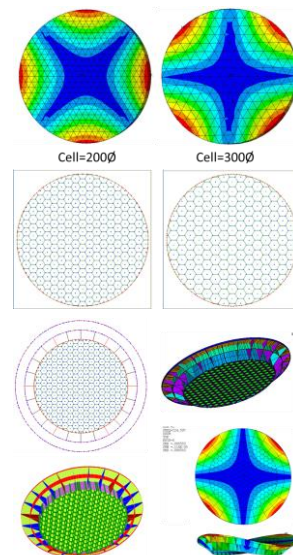
- First mode ~ 70 Hz.
- Mass ~ 2600 kg

#2: Hex Pockets with T-back

- First mode ~ 120 Hz.
- Mass ~ 1800 kg

#3: SOFIA Style

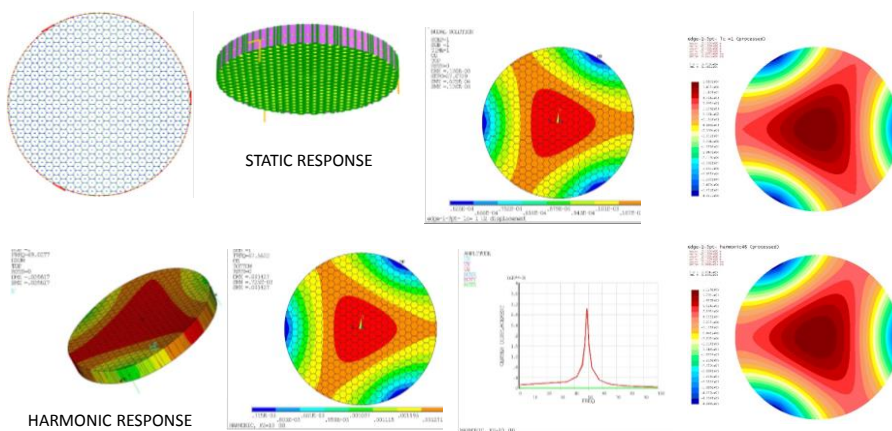
- First mode ~ 125 Hz.
- Mass ~ 1350 kg



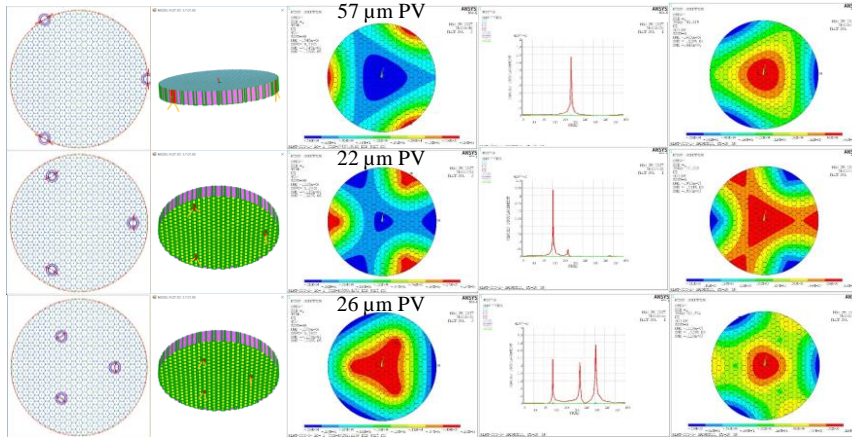
ZERODUR PM Trade Study: Multiple Designs

f	Depth	cell	width	mass	1 st bend	2 nd bend														f	depth	cell	width	mass	1 st bend	2 nd bend																				
2.5	250	173	25	2213	71	122														2.0	295	173	25	2748	85	139																				
2.5	250	173	50	2893	69	119														2.0	295	173	50	3423	87	141																				
2.5	f	depth	cell	width	mass	1 st bend	2 nd bend														2.0	f	depth	cell	width	mass	1 st bend	2 nd bend																		
f	2.5	310	173	25	3471	75	122														3.0	337	173	25	4318	86	136																			
2.5	250	173	50	3113	71	119														f	2.5	310	173	50	4353	83	144																			
1.5	250	173	125	008	310	005	511	41	65														3	337	173	125	008	310	010	680	42	70														
1.5	f	depth	cell	width	mass	1 st f	2 nd f														de	f	depth	cell	width	mass	1 st f	2 nd f																		
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Support trade studies



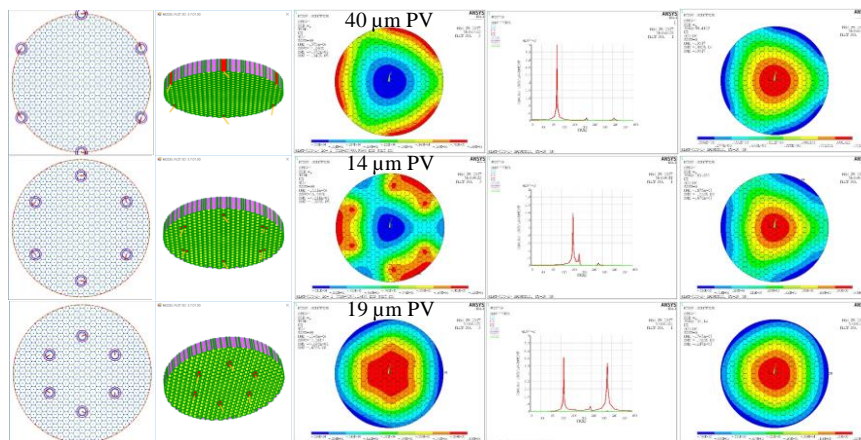
FLAT 3PT TRADE STUDY



Suspension system consists of beam elements with the desired stiffness and geometry.

Mirror and Mount system assumes 0.5% dampening give a Q (amplification factor) about 100X on transmissibility.

FLAT 6PT TRADE STUDY



UTAS 4-m Zerodur Mirror Design

Milled Open-Back Zerodur Mirror

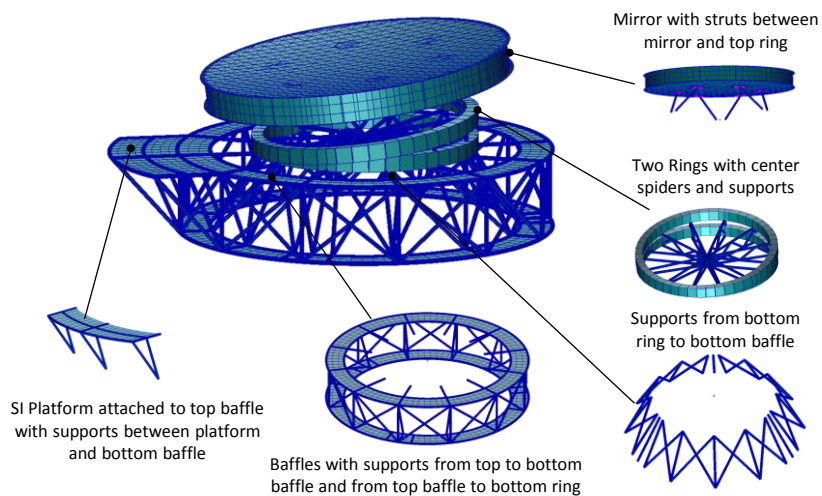
- Diameter 4.2 meters
- Mass 1200 kg
- First Mode 120 Hz Mounted



Primary Mirror Assembly

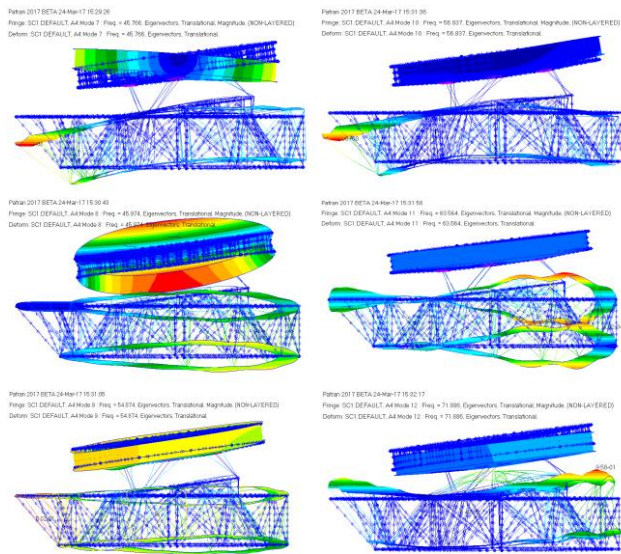
Mirror Substrate is 180 Hz.

Assembly is 45 Hz before attachment to Secondary Tower



PMA Modal Response

Mode 7 : Freq. = 45.766
 Mode 8 : Freq. = 45.974
 Mode 9 : Freq. = 54.874
 Mode 10 : Freq. = 56.837
 Mode 11 : Freq. = 63.564
 Mode 12 : Freq. = 71.886



Primary Mirror Dynamic WFE

Primary Mirror Dynamic Trefoil Specification is defined by
 Coronagraph Contrast Leakage Sensitivity

PV Trefoil Tolerance for Primary Mirror					
	from 1.5 to 2.5 λ/D	from 2.5 to 3.5 λ/D			
Alignment	4 th Order Radial	VVC6	VVC8	VVC10	Units
Trefoil	6 PV	0.0065 RMS	0.35 RMS	0.71 RMS	nanometers
		0.0184 PV	0.99 PV	2.008 PV	nanometers

We have not yet analyzed the dynamic WFE for our primary mirror candidates.

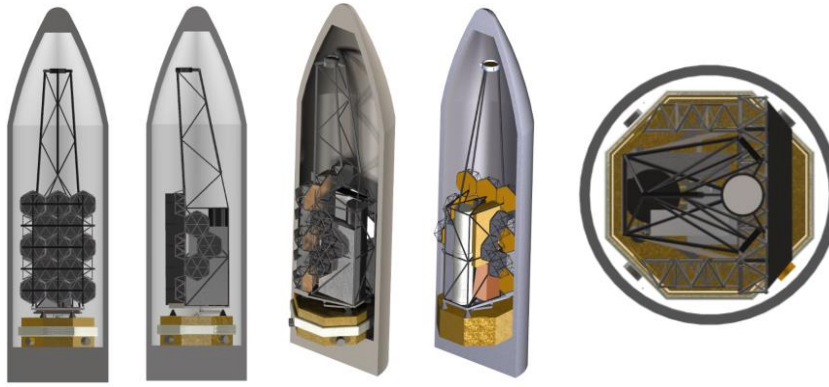
6.5-meter Segmented Concept

Concept ONLY
No Analysis

Conceptual Deployment Movie



Fairing Packaging



Deployed

